

**Warsaw University
of Technology**



**Faculty of Power and
Aeronautical Engineering**

WARSAW UNIVERSITY OF TECHNOLOGY

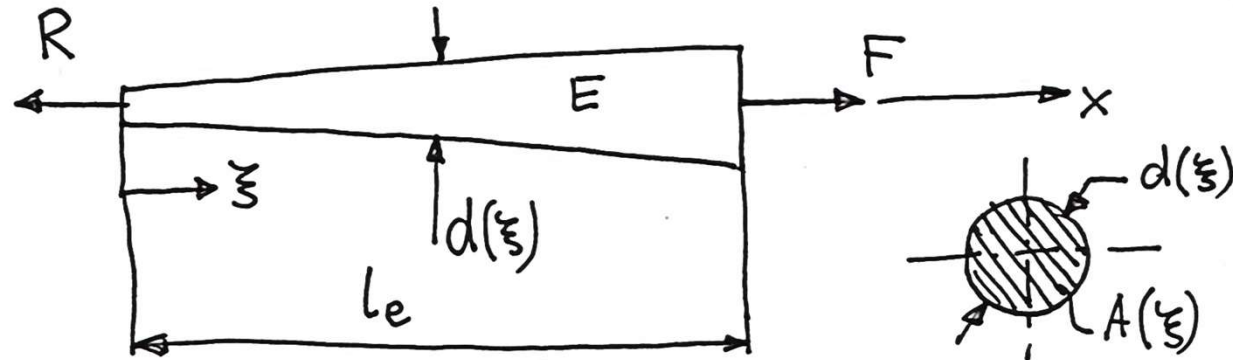
Institute of Aeronautics and Applied Mechanics

Finite element method (FEM)

Conical bar element

05.2021

CONICAL BAR ELEMENT WITH A CIRCULAR CROSS-SECTION



DIAMETER $d(\xi) = \frac{d_3 - d_1}{l_e} \cdot \xi + d_1$ (LINEAR FUNCTION OF COORD. ξ)

$d_1 = d(0)$, $d_3 = d(l_e)$

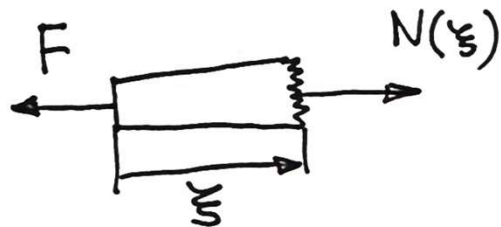
CROSS-SECTIONAL AREA : $A(\xi) = \frac{\pi d(\xi)^2}{4} = a\xi^2 + b\xi + c$ where:

$a = \frac{\pi (d_3 - d_1)^2}{4 l_e^2}$, $b = \frac{\pi (d_3 - d_1) d_1}{2 l_e}$, $c = \frac{\pi d_1^2}{4}$

ANALYTICAL SOLUTION

$$\sum F_x = 0 : -R + F = 0$$

$$R = F$$



INTERNAL FORCE:

$$N(\xi) - F = 0 \Rightarrow N(\xi) = F = \text{const.}$$

STRESS :
$$\sigma_x(\xi) = \frac{N(\xi)}{A(\xi)} = \frac{F}{A(\xi)}$$

STRAIN :
$$\epsilon_x(\xi) = \frac{\sigma_x(\xi)}{E} = \frac{F}{E \cdot A(\xi)}$$

DISPLACEMENT :

$$u(\xi) = \int_0^{\xi} \frac{F}{E A(x)} dx = \frac{F}{E} \int_0^{\xi} \frac{dx}{ax^2 + bx + c}$$

$$\Delta = b^2 - 4ac = \frac{\pi^2 (d_3 - d_1)^2 d_1^2}{4l^2} - 4 \cdot \frac{\pi (d_3 - d_1)^2}{4l^2} \cdot \frac{\pi d_1^2}{4} = 0$$

$$x_0 = -\frac{b}{2a} = -\frac{\pi (d_3 - d_1) d_1 \cdot 4l^2}{2l \cdot 2 \cdot \pi (d_3 - d_1)^2} = -\frac{l \cdot d_1}{d_3 - d_1}$$

$$\Rightarrow ax^2 + bx + c = a(x - x_0)^2$$

$$u(\xi) = \frac{F}{E} \int_0^{\xi} \frac{dx}{a(x - x_0)^2} = \left. \begin{array}{l} x - x_0 = y \\ dx = dy \\ x = 0 \Rightarrow y = -x_0 \\ x = \xi \Rightarrow y = \xi - x_0 \end{array} \right| = \frac{F}{Ea} \int_{-x_0}^{\xi - x_0} \frac{dy}{y^2} =$$

$$= \frac{F}{Ea} \left(\frac{y^{-2+1}}{-2+1} \right) \Big|_{-x_0}^{\xi - x_0} = -\frac{F}{Ea} \cdot \left(\frac{1}{y} \right) \Big|_{-x_0}^{\xi - x_0} =$$

$$= -\frac{F}{Ea} \left(\frac{1}{\xi - x_0} + \frac{1}{x_0} \right) = -\frac{F}{Ea} \left(\frac{\xi - x_0 + x_0}{(\xi - x_0)x_0} \right) =$$

$$\begin{aligned}
 = u(\xi) &= -\frac{F\xi}{Ea} \cdot \frac{1}{\left(\xi + \frac{le \cdot d_1}{d_3 - d_1}\right) \cdot \left(-\frac{le d_1}{(d_3 - d_1)}\right)} = \frac{F\xi}{Ea} \cdot \frac{1}{\frac{(\xi \cdot (d_3 - d_1) + le d_1) le d_1}{(d_3 - d_1)^2}} = \\
 &= \frac{F\xi (d_3 - d_1)^2 \cdot 4le^2}{E \cdot \pi (d_3 - d_1)^2 \cdot (\xi (d_3 - d_1) + le d_1) le d_1} = \frac{4Fle\xi}{\pi E d_1 (\xi (d_3 - d_1) + le d_1)}
 \end{aligned}$$

$$u(0) = 0,$$

$$u\left(\frac{le}{2}\right) = \frac{4Fle \frac{le}{2}}{\pi E d_1 \left(\frac{le}{2}(d_3 - d_1) + le d_1\right)} = \frac{2Fle^2}{\pi E d_1 le \left(\frac{d_3 + d_1}{2}\right)} = \frac{4Fle}{\pi E d_1 (d_1 + d_3)},$$

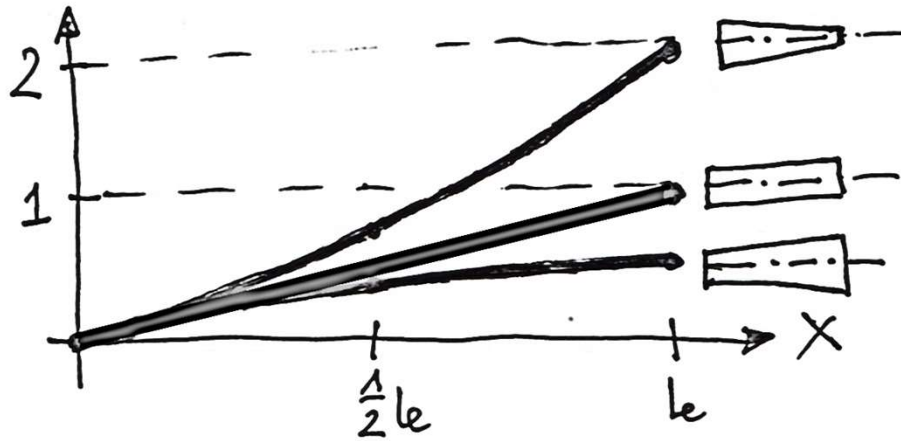
$$u(le) = \frac{4Fle^2}{\pi E d_1 (le(d_3 - d_1) + le d_1)} = \frac{4Fle}{\pi E d_1 d_3}$$

$\frac{d_3}{d_1}$	$u(\frac{l}{2}) / \frac{Fl_e}{EA_1}$	$u(l) / \frac{Fl_e}{EA_1}$
$\frac{1}{2}$	$\frac{2}{3}$	2
1	$\frac{1}{2}$	1
2	$\frac{1}{3}$	$\frac{1}{2}$

$$u(\xi) = \frac{4Fl_e \xi}{\pi E d_1 (\xi(d_3 - d_1) + l d_1)}$$

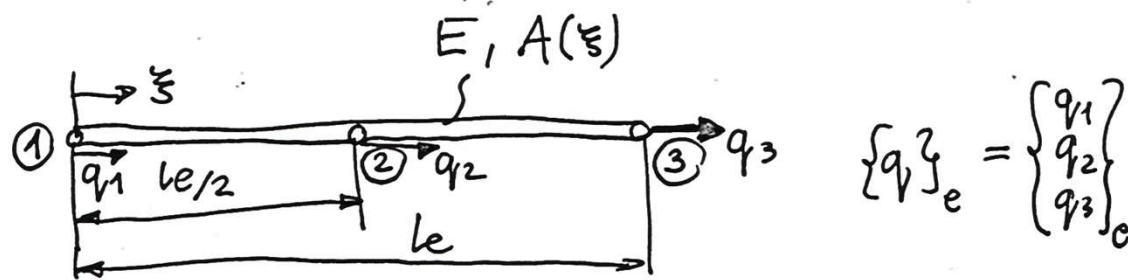
$$A_1 = \frac{\pi d_1^2}{4}$$

$$u / \left(\frac{Fl_e}{EA_1} \right)$$



$\Rightarrow u(\xi)$ APPROXIMATION
BY PARABOLA

FINITE ELEMENT



ASSUMPTION : MIDSIDE NODE ② at $l/2$.

$$u(\xi) = \alpha_1 + \alpha_2 \cdot \xi + \alpha_3 \cdot \xi^2 = 1 \cdot \alpha_1 + \xi \cdot \alpha_2 + \xi^2 \cdot \alpha_3 = [1, \xi, \xi^2] \cdot \begin{Bmatrix} \alpha_1 \\ \alpha_2 \\ \alpha_3 \end{Bmatrix}$$

$$u(0) = q_1 \Rightarrow q_1 = 1 \cdot \alpha_1 + 0 \cdot \alpha_2 + 0 \cdot \alpha_3$$

$$u\left(\frac{l}{2}\right) = q_2 \Rightarrow q_2 = 1 \cdot \alpha_1 + \frac{l}{2} \cdot \alpha_2 + \frac{l^2}{4} \cdot \alpha_3$$

$$u(l) = q_3 \Rightarrow q_3 = 1 \cdot \alpha_1 + l \cdot \alpha_2 + l^2 \cdot \alpha_3$$

$$\begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix}_{3 \times 1} = \begin{bmatrix} 1 & 0 & 0 \\ 1 & \frac{l}{2} & \frac{l^2}{4} \\ 1 & l & l^2 \end{bmatrix} \cdot \begin{Bmatrix} \alpha_1 \\ \alpha_2 \\ \alpha_3 \end{Bmatrix} = [A]_{3 \times 3} \cdot \{\alpha\}_{3 \times 1}, \quad \{\alpha\}_{3 \times 1} = [A^{-1}]_{3 \times 3} \cdot \{q\}_e_{3 \times 1}$$

$$[A]_{3 \times 3} = \begin{bmatrix} 1 & 0 & 0 \\ 1 & \frac{le}{2} & \frac{le^2}{4} \\ 1 & le & le^2 \end{bmatrix}$$

$$, \quad \det [A] = \frac{le}{2} \cdot le^2 - \frac{le^2}{4} \cdot le = \frac{le^3}{4}$$

$$C_{11} = \frac{le}{2} \cdot le^2 - \frac{le^2}{4} \cdot le = \frac{le^3}{4} ,$$

$$C_{12} = -(1 \cdot le^2 - \frac{le^2}{4} \cdot 1) = -\frac{3}{4} le^2$$

$$C_{13} = 1 \cdot le - \frac{le}{2} \cdot 1 = \frac{le}{2} ,$$

$$C_{21} = -(0 \cdot le^2 - 0 \cdot le) = 0$$

$$C_{22} = 1 \cdot le^2 - 0 \cdot 1 = le^2 ,$$

$$C_{23} = -(1 \cdot le - 0 \cdot 1) = -le$$

$$C_{31} = 0 \cdot \frac{le^2}{4} - 0 \cdot \frac{le}{2} = 0 ,$$

$$C_{32} = -(1 \cdot \frac{le^2}{4} - 0 \cdot 1) = -\frac{le^2}{4}$$

$$C_{33} = 1 \cdot \frac{le}{2} - 0 \cdot 1 = \frac{le}{2} .$$

$$[A^c] = \begin{bmatrix} C_{11} & C_{12} & C_{13} \\ C_{21} & C_{22} & C_{23} \\ C_{31} & C_{32} & C_{33} \end{bmatrix} = \frac{1}{4} \begin{bmatrix} le^3 & -3le^2 & 2le \\ 0 & 4le^2 & -4le \\ 0 & -le^2 & 2le \end{bmatrix}$$

$$[A^c]^T = \frac{1}{4} \begin{bmatrix} le^3 & 0 & 0 \\ -3le^2 & 4le^2 & -le^2 \\ 2le & -4le & 2le \end{bmatrix}$$

$$[A^{-1}] = \frac{1}{\det[A]} \cdot [A^c]^T = \frac{4}{le^3} \cdot [A^c]^T = \begin{bmatrix} 1 & 0 & 0 \\ -\frac{3}{le} & \frac{4}{le} & -\frac{1}{le} \\ \frac{2}{le^2} & -\frac{4}{le^2} & \frac{2}{le^2} \end{bmatrix}$$

NODAL APPROXIMATION / SHAPE FUNCTIONS

$$\begin{aligned}
 u(\xi) &= \underbrace{[1, \xi, \xi^2]}_{3 \times 1} \cdot \underbrace{\{\alpha\}}_{3 \times 1} = \underbrace{[1, \xi, \xi^2]}_{3 \times 3} \cdot \underbrace{[A^{-1}]}_{3 \times 3} \cdot \underbrace{\{q\}_e}_{3 \times 1} = \\
 &= \underbrace{[N_1(\xi), N_2(\xi), N_3(\xi)]}_{1 \times 3} \cdot \underbrace{\{q\}_e}_{3 \times 1} = \underbrace{[N(\xi)]}_{1 \times 3} \cdot \underbrace{\{q\}_e}_{3 \times 1}
 \end{aligned}$$

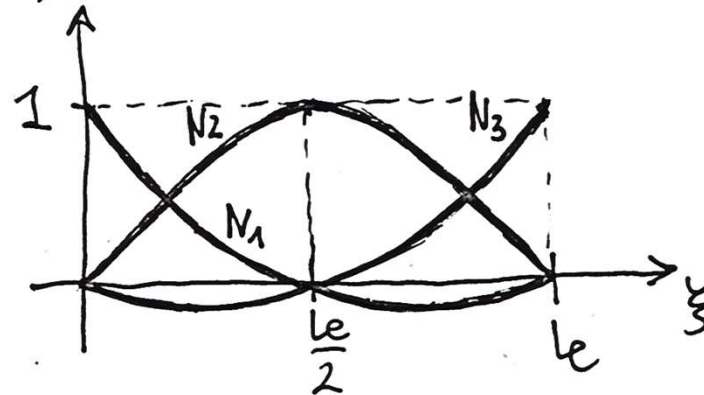
where :

$$N_1(\xi) = \frac{2\xi^2}{l_e^2} - \frac{3\xi}{l_e} + 1$$

$$N_2(\xi) = -\frac{4\xi^2}{l_e^2} + \frac{4\xi}{l_e}$$

$$N_3(\xi) = \frac{2\xi^2}{l_e^2} - \frac{\xi}{l_e}$$

$N_i(\xi)$



$$\frac{dN_1}{d\xi} = \frac{4}{l_e} \left(\frac{\xi}{l_e} - \frac{3}{4} \right), \quad \frac{dN_2}{d\xi} = \frac{4}{l_e} \left(1 - \frac{2\xi}{l_e} \right), \quad \frac{dN_3}{d\xi} = \frac{4}{l_e} \left(\frac{\xi}{l_e} - \frac{1}{4} \right)$$

$$\varepsilon_x(\xi) = \frac{du}{d\xi} = \underbrace{L}_{1 \times 3} \underbrace{\frac{dN}{d\xi}}_{3 \times 1} \cdot \underbrace{\{q\}}_{3 \times 1} \quad \text{or:} \quad \varepsilon_x(\xi) = \underbrace{Lq}_{1 \times 3} \cdot \underbrace{\left\{ \frac{dN}{d\xi} \right\}}_{3 \times 1}$$

$$U_e = \frac{1}{2} \int_{\Omega_e} \underbrace{L}_{1 \times 6} \underbrace{\varepsilon}_{6 \times 1} \cdot \underbrace{\{\sigma\}}_{6 \times 1} d\Omega_e = \frac{1}{2} \int_{\Omega_e} \varepsilon_x \cdot \sigma_x d\Omega_e = \frac{1}{2} \int_{\Omega_e} \varepsilon_x \cdot E \cdot \varepsilon_x d\Omega_e =$$

$$= \frac{1}{2} \int_{\Omega_e} \underbrace{Lq}_{1 \times 3} \cdot \underbrace{\left\{ \frac{dN}{d\xi} \right\}}_{3 \times 1} \cdot E \cdot \underbrace{L}_{1 \times 3} \cdot \underbrace{\{q\}}_{3 \times 1} d\Omega_e = \left(\begin{array}{l} d\Omega_e = A(\xi) \cdot d\xi \\ A(\xi) = a\xi^2 + b\xi + c \end{array} \right) =$$

$$= \frac{1}{2} \underbrace{Lq}_{1 \times 3} \cdot E \cdot \int_0^{l_e} \underbrace{\left\{ \frac{dN}{d\xi} \right\}}_{3 \times 1} \cdot \underbrace{L}_{1 \times 3} \cdot A(\xi) d\xi \cdot \underbrace{\{q\}}_{3 \times 1} = \frac{1}{2} \underbrace{Lq}_{1 \times 3} \cdot \underbrace{[k]}_{3 \times 3} \cdot \underbrace{\{q\}}_{3 \times 1}$$

STIFFNESS MATRIX

$$\underset{3 \times 3}{[k]}_e = E \cdot \left[\begin{array}{ccc} \int_0^{le} N_1' N_1' A(\xi) d\xi & \int_0^{le} N_1' N_2' A(\xi) d\xi & \int_0^{le} N_1' N_3' A(\xi) d\xi \\ \int_0^{le} N_2' N_1' A(\xi) d\xi & \int_0^{le} N_2' N_2' A(\xi) d\xi & \int_0^{le} N_2' N_3' A(\xi) d\xi \\ \int_0^{le} N_3' N_1' A(\xi) d\xi & \int_0^{le} N_3' N_2' A(\xi) d\xi & \int_0^{le} N_3' N_3' A(\xi) d\xi \end{array} \right]$$

$$k_{11} = E \int_0^l \frac{dN_1}{d\xi} \cdot \frac{dN_1}{d\xi} \cdot A(\xi) d\xi = E \int_0^l \frac{4}{l} \cdot \frac{4}{l} \cdot \left(\frac{\xi}{l} - \frac{3}{4}\right)^2 \cdot (a\xi^2 + b\xi + c) d\xi =$$

$$= \frac{16E}{l^2} \cdot \int_0^l \left(\frac{\xi^2}{l^2} - 2 \cdot \frac{\xi}{l} \cdot \frac{3}{4} + \frac{9}{16}\right)^2 (a\xi^2 + b\xi + c) d\xi =$$

$$= \frac{16E}{l^2} \int_0^l \left(\frac{a}{l^2} \xi^4 + \frac{b}{l^2} \xi^3 + \frac{c}{l^2} \xi^2 - \frac{3a}{2l} \xi^3 - \frac{3b}{2l} \xi^2 - \frac{3c}{2l} \cdot \xi +$$

$$+ \frac{9a}{16} \xi^2 + \frac{9b}{16} \xi + \frac{9c}{16}\right) d\xi =$$

$$\begin{aligned}
= k_{11} &= \frac{16E}{l^2} \int_0^l \left(\frac{a}{l^2} \xi^4 + \left(\frac{b}{l^2} - \frac{3a}{2l} \right) \xi^3 + \left(\frac{c}{l^2} - \frac{3b}{2l} + \frac{9a}{16} \right) \xi^2 + \left(\frac{9b}{16} - \frac{3c}{2l} \right) \xi + \frac{9c}{16} \right) d\xi = \\
&= \frac{16E}{l^2} \left(\frac{a\xi^5}{5l^2} + \left(\frac{b}{4l^2} - \frac{3a}{8l} \right) \xi^4 + \left(\frac{c}{3l^2} - \frac{b}{2l} + \frac{3a}{16} \right) \xi^3 + \left(\frac{9b}{32} - \frac{3c}{4l} \right) \xi^2 + \frac{9c\xi}{16} \right) \Big|_0^l = \\
&= \frac{16E}{l^2} \left(\frac{al^3}{5} + \frac{bl^2}{4} - \frac{3ale^3}{8} + \frac{cl^2}{3} - \frac{6l^2}{2} + \frac{3ale^3}{16} + \frac{9bl^2}{32} - \frac{3cl^2}{4} + \frac{9cl^2}{16} \right) = \\
&= 16E \left(\left(\frac{l}{5} - \frac{3l}{8} + \frac{3l}{16} \right) a + \left(\frac{1}{4} - \frac{1}{2} + \frac{9}{32} \right) \cdot b + \left(\frac{1}{3l} - \frac{3}{4l} + \frac{9}{16l} \right) \cdot c \right) = \\
&= 16E \left(\left(\frac{16-30+15}{80} \right) ale + \left(\frac{8-16+9}{32} \right) b + \left(\frac{16-36+27}{48l} \right) c \right) = \\
&= 16E \left(\frac{ale}{80} + \frac{b}{32} + \frac{7c}{48l} \right) = E \left(\frac{ale}{5} + \frac{b}{2} + \frac{7c}{3l} \right)
\end{aligned}$$

$$k_{12} = E \int_0^l \frac{dN_1}{d\xi} \cdot \frac{dN_2}{d\xi} \cdot A(\xi) d\xi = -E \left(\frac{2ale}{5} + \frac{2b}{3} + \frac{8c}{3le} \right) = k_{21}$$

$$k_{13} = E \int_0^l \frac{dN_1}{d\xi} \cdot \frac{dN_3}{d\xi} A(\xi) d\xi = E \left(\frac{ale}{5} + \frac{b}{6} + \frac{c}{3le} \right) = k_{31}$$

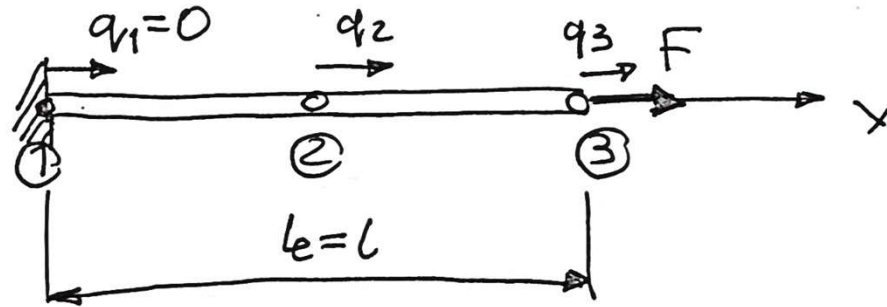
$$k_{22} = E \int_0^l \frac{dN_2}{d\xi} \cdot \frac{dN_2}{d\xi} A(\xi) d\xi = E \left(\frac{32ale}{15} + \frac{8b}{3} + \frac{16c}{3le} \right)$$

$$k_{23} = E \int_0^l \frac{dN_2}{d\xi} \cdot \frac{dN_3}{d\xi} A(\xi) d\xi = E \left(\frac{26ale}{15} + 2b + \frac{8c}{3le} \right) = k_{32}$$

$$k_{33} = E \int_0^l \frac{dN_3}{d\xi} \cdot \frac{dN_3}{d\xi} A(\xi) d\xi = E \left(\frac{23ale}{15} + \frac{11b}{6} + \frac{7c}{3le} \right)$$

$$[k]_e = \begin{bmatrix} k_{11} & k_{12} & k_{13} \\ k_{21} & k_{22} & k_{23} \\ k_{31} & k_{32} & k_{33} \end{bmatrix} \quad ; \quad \det [k]_e = 0$$

1°) 1 FE:



$$[K] = [k]_e, \quad \{q\} = \{q\}_e, \quad \{F\} = \begin{Bmatrix} R_1 \\ 0 \\ F \end{Bmatrix}, \quad q_1 = 0$$

3×3 3×3 , 3×1 3×1 , 3×1

$$\begin{bmatrix} k_{22} & k_{23} \\ k_{32} & k_{33} \end{bmatrix} \begin{Bmatrix} q_2 \\ q_3 \end{Bmatrix} = \begin{Bmatrix} 0 \\ F \end{Bmatrix}, \quad k_{32} = k_{23}$$

$$k_{22} q_2 + k_{23} q_3 = 0 \Rightarrow q_3 = -\frac{k_{22}}{k_{23}} q_2$$

$$k_{23} q_2 + k_{33} q_3 = F$$

$$k_{23} q_2 - \frac{k_{22}}{k_{23}} q_2 = F \Rightarrow \frac{k_{23}^2 - k_{22}}{k_{23}} q_2 = F$$

SOLUTION

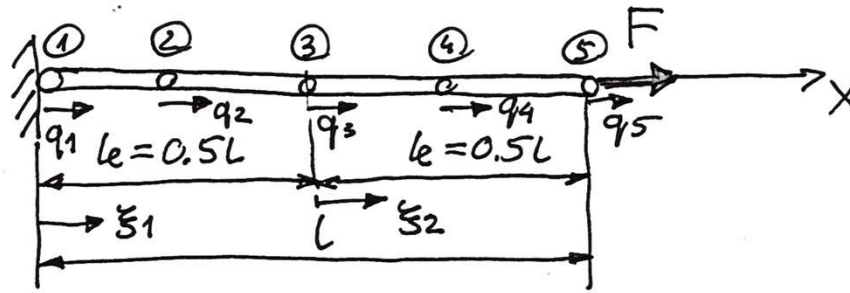
$$q_2 = \frac{F k_{23}}{k_{23}^2 - k_{22}} \quad , \quad q_3 = -\frac{F k_{22}}{k_{23}^2 - k_{22}}$$

$$u(\xi) = \underset{1 \times 3}{[N(\xi)]} \cdot \underset{3 \times 1}{\{q\}_1} = [N_1(\xi), N_2(\xi), N_3(\xi)] \cdot \begin{Bmatrix} 0 \\ q_2 \\ q_3 \end{Bmatrix}_1 = N_2(\xi) \cdot q_2 + N_3(\xi) \cdot q_3$$

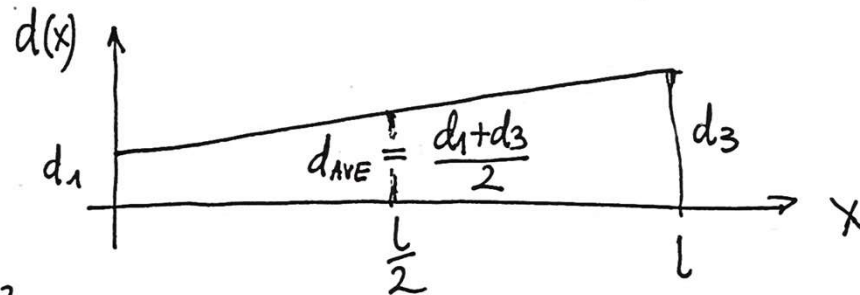
$$\varepsilon_x(\xi) = \frac{dN_2}{d\xi} \cdot q_2 + \frac{dN_3}{d\xi} \cdot q_3$$

$$\sigma_x(\xi) = E \cdot \varepsilon_x(\xi) \quad , \quad N(\xi) = \sigma_x(\xi) \cdot A(\xi)$$

2°) 2 FEs:



$$\{q\}_1 = \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \end{Bmatrix}, \quad \{q\}_2 = \begin{Bmatrix} q_3 \\ q_4 \\ q_5 \end{Bmatrix}, \quad \{q\}_{5 \times 1} = \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \end{Bmatrix}$$



$$A_1(\xi_1) = a_1 \xi_1^2 + b_1 \xi_1 + c_1$$

$$a_1 = \frac{\pi (d_{AVE} - d_1)^2}{4l^2}, \quad b_1 = \frac{\pi (d_{AVE} - d_1) d_1}{2le}, \quad c_1 = \frac{\pi d_1^2}{4}$$

$$A_2(\xi_2) = a_2 \xi_2^2 + b_2 \xi_2 + c_2$$

$$a_2 = \frac{\pi (d_3 - d_{AVE})^2}{4l^2}, \quad b_2 = \frac{\pi (d_3 - d_{AVE}) d_{AVE}}{2le}, \quad c_2 = \frac{\pi d_{AVE}^2}{4}$$

STIFFNESS MATRIX / SOLUTION

$$\begin{matrix} [K] \\ 5 \times 5 \end{matrix} = \begin{bmatrix} \text{[k]}_1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \text{[k]}_2 \end{bmatrix}, \quad \begin{matrix} \{q\} \\ 5 \times 1 \end{matrix} = \begin{Bmatrix} q_1 \\ q_2 \\ q_3 \\ q_4 \\ q_5 \end{Bmatrix}, \quad \begin{matrix} \{F\} \\ 5 \times 1 \end{matrix} = \begin{Bmatrix} R_1 \\ 0 \\ 0 \\ 0 \\ F \end{Bmatrix}; \quad q_1 = 0$$

$$\begin{bmatrix} \text{[k]}_1 & 0 & 0 \\ 0 & 0 & 0 \\ 0 & 0 & \text{[k]}_2 \end{bmatrix} \cdot \begin{Bmatrix} q_2 \\ q_3 \\ q_4 \\ q_5 \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \\ 0 \\ F \end{Bmatrix} \Rightarrow \begin{Bmatrix} q_2 \\ q_3 \\ q_4 \\ q_5 \end{Bmatrix}$$

DISPLACEMENT :

$$u_1(\xi_1) = \underset{1 \times 3}{[N(\xi_1)]} \cdot \underset{3 \times 1}{\{q\}_1}, \quad u_2(\xi_2) = \underset{1 \times 3}{[N(\xi_2)]} \cdot \underset{3 \times 1}{\{q\}_2}$$

ELEMENT SOLUTION

STRAIN :

$$\epsilon_{x_1}(\xi_1) = \underset{1 \times 3}{L \frac{dN}{d\xi_1}} \cdot \{q\}_1, \quad \epsilon_{x_2}(\xi_2) = \underset{1 \times 3}{L \frac{dN}{d\xi_2}} \cdot \{q\}_2$$

STRESS :

$$\sigma_{x_1}(\xi_1) = E \cdot \epsilon_{x_1}(\xi_1), \quad \sigma_{x_2}(\xi_2) = E \cdot \epsilon_{x_2}(\xi_2)$$

INTERNAL FORCES :

$$N_1(\xi_1) = \sigma_{x_1}(\xi_1) \cdot A_1(\xi_1), \quad N_2(\xi_2) = \sigma_{x_2}(\xi_2) \cdot A_2(\xi_2)$$

ELEMENT SOLUTION

STRAIN :

$$\epsilon_{x_1}(\xi_1) = \underset{1 \times 3}{L \frac{dN}{d\xi_1}} \cdot \{q\}_1, \quad \epsilon_{x_2}(\xi_2) = \underset{1 \times 3}{L \frac{dN}{d\xi_2}} \cdot \{q\}_2$$

STRESS :

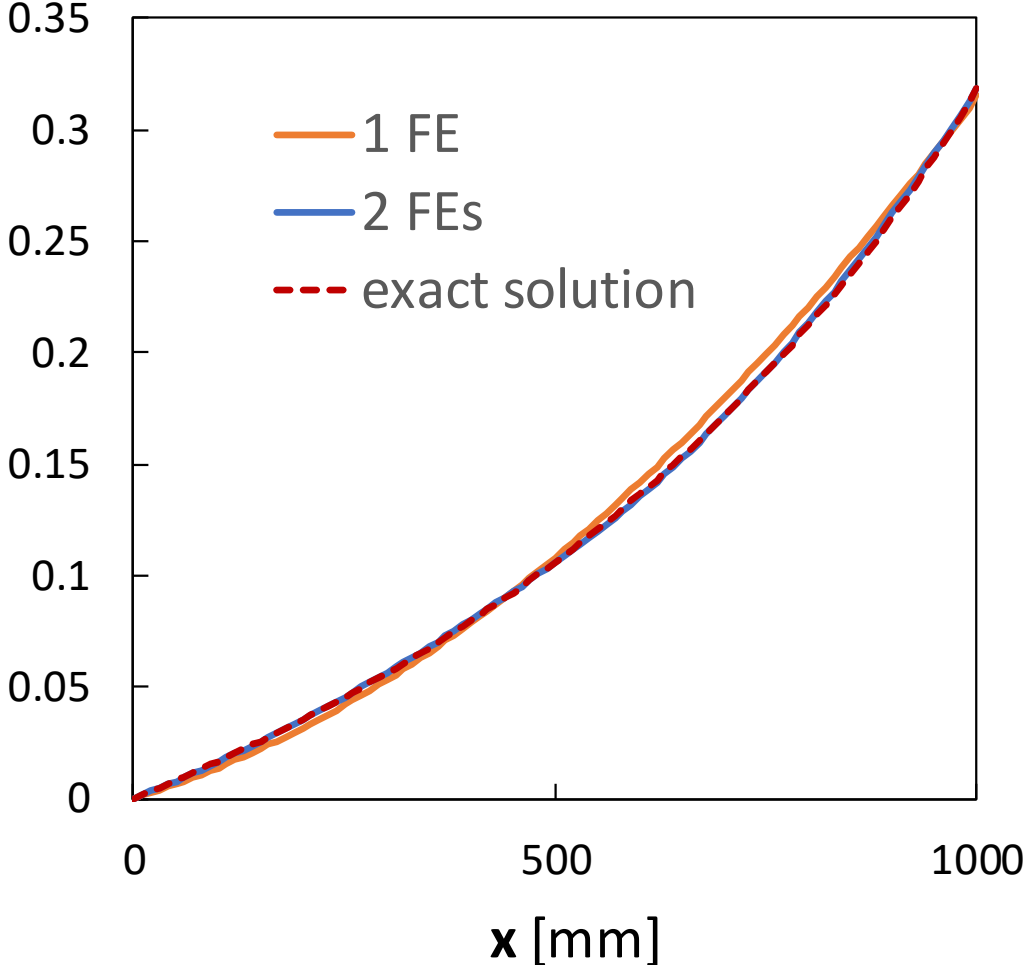
$$\sigma_{x_1}(\xi_1) = E \cdot \epsilon_{x_1}(\xi_1), \quad \sigma_{x_2}(\xi_2) = E \cdot \epsilon_{x_2}(\xi_2)$$

INTERNAL FORCES :

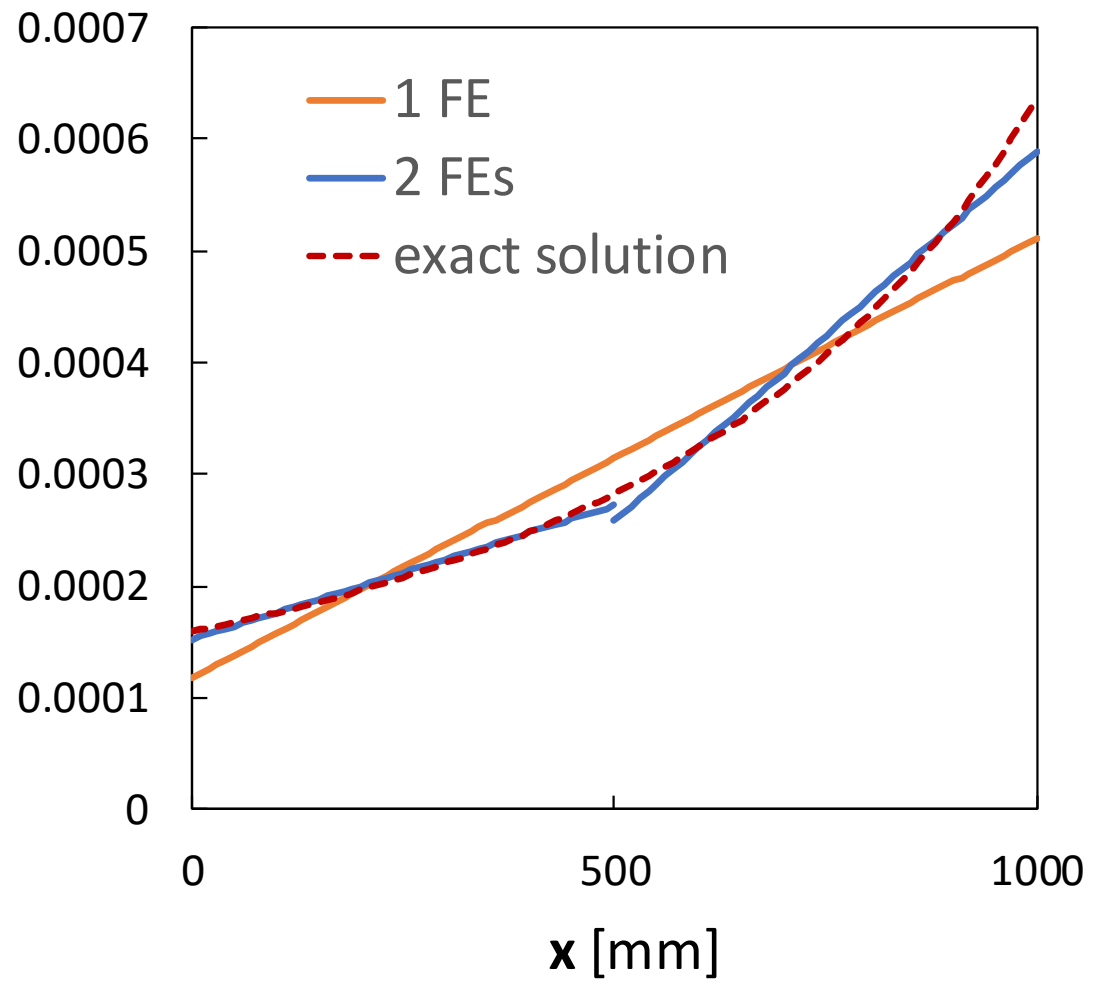
$$N_1(\xi_1) = \sigma_{x_1}(\xi_1) \cdot A_1(\xi_1), \quad N_2(\xi_2) = \sigma_{x_2}(\xi_2) \cdot A_2(\xi_2)$$

displacement u [mm]

$E = 2 \cdot 10^5 \text{ MPa}$
 $l = 1000 \text{ mm}$
 $d_1 = 20 \text{ mm}$
 $d_3 = 10 \text{ mm}$
 $F = 10000 \text{ N}$



strain



$$E = 2 \cdot 10^5 \text{ MPa}$$

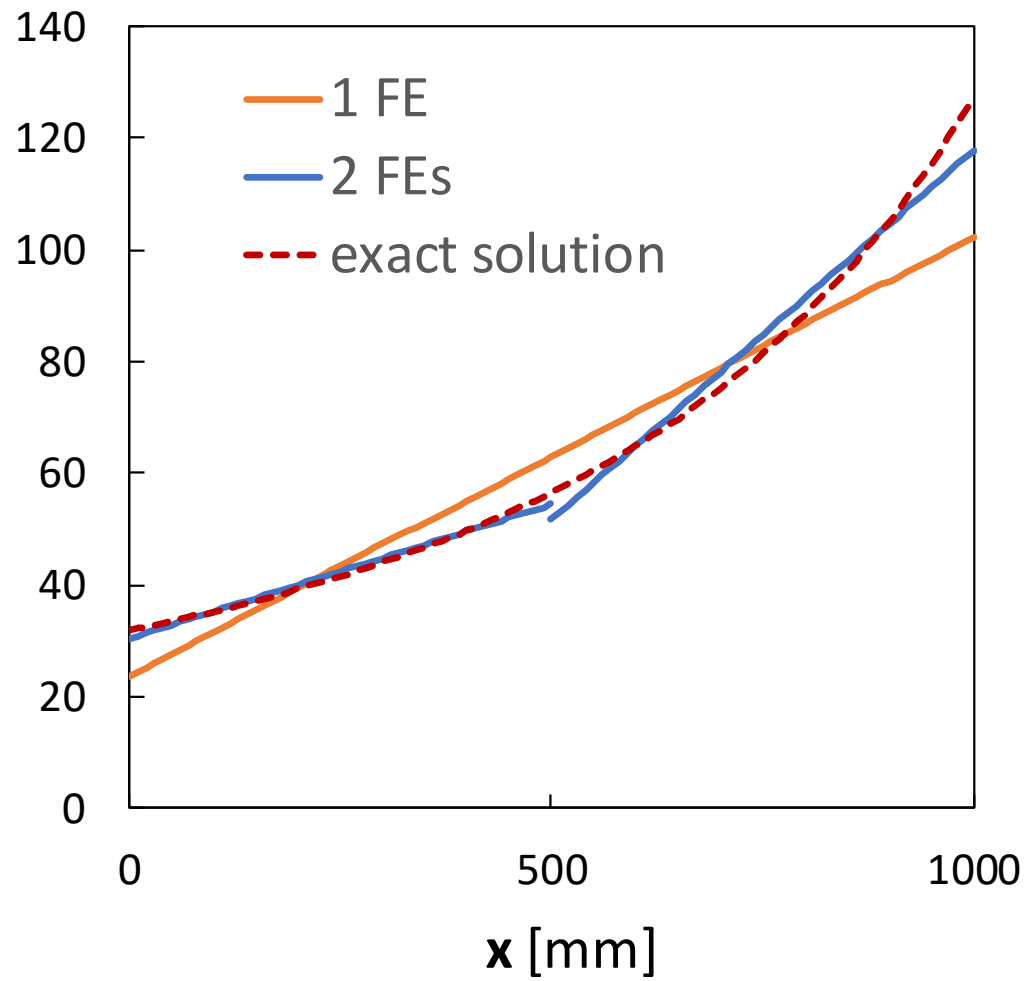
$$l = 1000 \text{ mm}$$

$$d_1 = 20 \text{ mm}$$

$$d_3 = 10 \text{ mm}$$

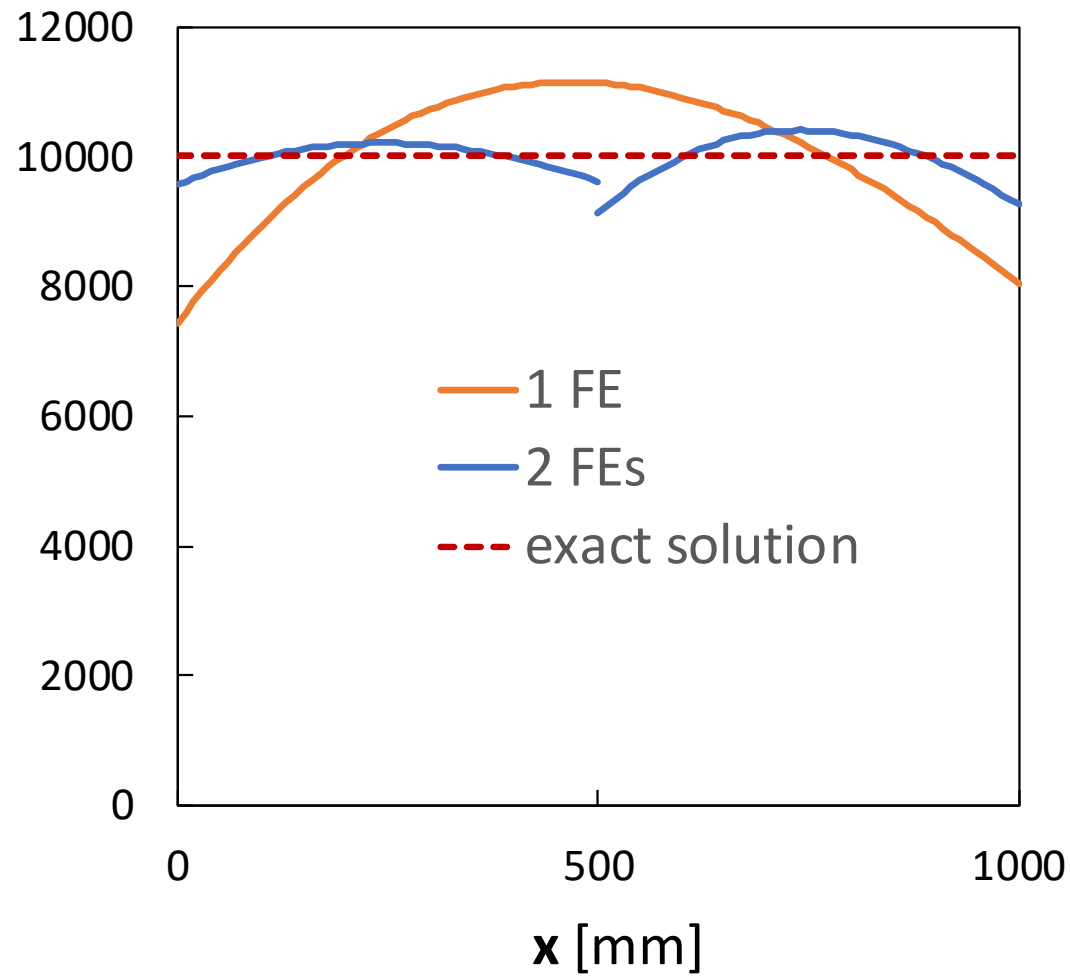
$$F = 10000 \text{ N}$$

stress [MPa]



$E = 2 \cdot 10^5 \text{ MPa}$
 $l = 1000 \text{ mm}$
 $d_1 = 20 \text{ mm}$
 $d_3 = 10 \text{ mm}$
 $F = 10000 \text{ N}$

internal force N [N]



$$E = 2 \cdot 10^5 \text{ MPa}$$

$$l = 1000 \text{ mm}$$

$$d_1 = 20 \text{ mm}$$

$$d_3 = 10 \text{ mm}$$

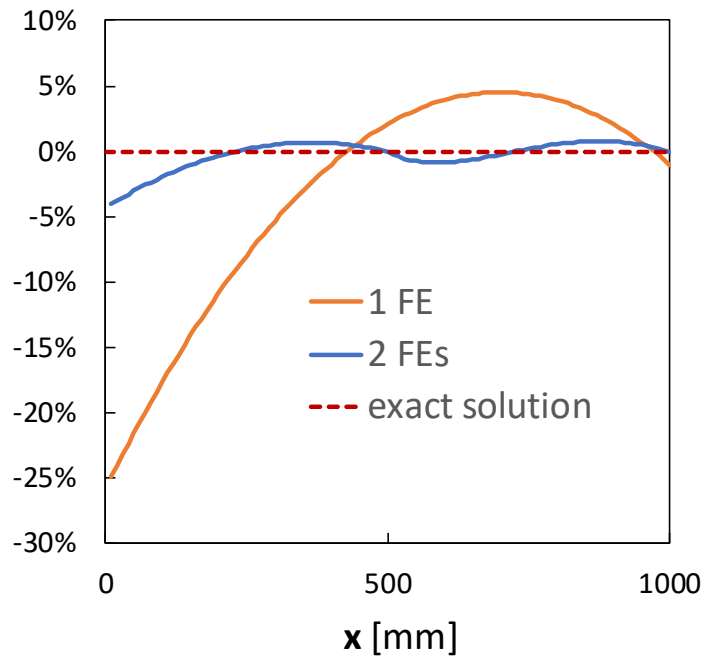
$$F = 10000 \text{ N}$$

reaction

$$R_1 = F$$

RELATIVE ERRORS

relative error of displacement u



relative error of strain, stress, and internal force

